

***Numerical Dynamics of a Cable Subjected to Frictional Impact***

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R 1.23 Tue Z3 11:20-11:30

The dynamics of an elastic cable subjected to frictional contact are investigated numerically. The talk will address the formulation of a Finite Element formulation for the finite displacement of one cable subjected to the presence of one obstacle.

The cable is a curvilinear domain which torsion and bending are negligible compared to the axial force, termed as tension. Moreover, the cable is a structure which cannot overcome compression. The position and velocity of the centerline are given at any time  $t$  through the relation  $S \rightarrow (q(S, t), v(S, t))$  where  $S$  is a curvilinear coordinate in the unstretched configuration. In the sequel, these dependencies are omitted for conciseness. We will refer to  $S$  and  $t$  differentiation with  $\bullet'$  and  $\bullet\dot{\phantom{\bullet}}$ . The local form of system equations [1, 2] are given as follows:

$$\rho\dot{v} + 2cv = \left( EA (\|q'\| - 1)' \frac{q'}{\|q'\|} \right) + f \quad \text{such that } \|q'\| - 1 \geq 0 \quad (1)$$

It will be explained how a discrete nonlinear problem is obtained to predict the unconstrained dynamics of the cable in the following matrix form via finite element method [3] and how the latter is reformulated into a measure differential inclusion

$$\mathbf{M}\dot{\mathbf{v}} + \mathbf{C}(\mathbf{q})\mathbf{v} + \mathbf{K}(\mathbf{q})\mathbf{q} = \mathbf{f} \rightsquigarrow \begin{cases} \mathbf{M}\dot{\mathbf{v}} + \mathbf{C}(\mathbf{q})\mathbf{v} + \mathbf{K}(\mathbf{q})\mathbf{q} = \mathbf{f} dt + d\mathbf{r} \\ \mathbf{v}^+ = \dot{\mathbf{q}}^+ \\ \mathbf{q}(t = 0) \in \mathcal{C}(t = 0) , \mathbf{v}(t = 0^-) = \dot{\mathbf{q}}_0 \end{cases} \quad (2)$$

where  $\mathcal{C}(t)$  is defined as the subspace where the dynamics are constrained to evolve, here due to the presence of one obstacle, which reads

$$\mathcal{C}(t) = \left\{ \mathbf{q} \in \mathbb{R}^d , \mathbf{g}(\mathbf{q}) \geq \mathbf{0} \text{ (vector inequality)} \right\} \quad (3)$$

The measure differential inclusion will be used to derive a time-stepping scheme called the NonSmooth Contact Dynamics methods [4]. The latter relies on an implicit scheme of low order which can handle jumps at the

velocity level. A restitution coefficient,  $e$ , links the velocity before and after impact and the Coulomb law is used to model the friction ( $\mu$  Coulomb coefficient). At each time step, the frictional contact problem is attacked via Lemke Method [5]. It will be explained how the local coordinates at contacting points yields to solve the Linear Complementarity Problem just below

$$\left\{ \begin{array}{l}
 \mathbf{v}^{k+1} = \mathbf{v}^f + \widehat{\mathbf{M}}^{-1} \left( \mathbf{H}_N^\top \mathbf{r}^1 + \mathbf{H}_T^\top (\mu \mathbf{r}^1 - \mathbf{r}^2) \right) \quad , \quad \widehat{\mathbf{M}} = \mathbf{M} + h\mathbf{C} + h^2\Delta\mathbf{K} \\
 \mathbf{u}_N^1 = \mathbf{H}_N \mathbf{v}^f \quad , \quad \mathbf{u}_N^0 = \mathbf{H}_N \mathbf{v}^k \quad , \quad \mathbf{u}_T^1 = \mathbf{H}_T \mathbf{v}^f \quad , \quad \mathbf{u}_T^0 = \mathbf{H}_T \mathbf{v}^k \\
 \text{For all } \alpha \text{ such that } \left( \mathbf{g}(\mathbf{q}^k + h\mathbf{v}^f) \right)_\alpha \leq 0 : \\
 \mathbf{0} \leq \begin{bmatrix} \widehat{\mathbf{W}}_{NN} + \mu \widehat{\mathbf{W}}_{NT} & -\widehat{\mathbf{W}}_{NT} & \mathbf{0} \\ -\widehat{\mathbf{W}}_{TN} - \mu \widehat{\mathbf{W}}_{TT} & \widehat{\mathbf{W}}_{TT} & \mathbf{I} \\ 2\mu \mathbf{I} & -\mathbf{I} & \mathbf{0} \end{bmatrix} \begin{bmatrix} \mathbf{r}_\alpha^1 \\ \mathbf{r}_\alpha^2 \\ (\mathbf{u}_T^1)^+ \end{bmatrix} + \begin{bmatrix} (\mathbf{u}_N^1)_\alpha + e (\mathbf{u}_N^0)_\alpha \\ -(\mathbf{u}_T^1)_\alpha \\ \mathbf{0} \end{bmatrix} \perp \begin{bmatrix} \mathbf{r}_\alpha^1 \\ \mathbf{r}_\alpha^2 \\ (\mathbf{u}_T^1)^+ \end{bmatrix} \geq \mathbf{0} \\
 \text{For all } \alpha \text{ such that } \left( \mathbf{g}(\mathbf{q}^k + h\mathbf{v}^f) \right)_\alpha > 0 : \\
 \mathbf{r}_\alpha^1 = \mathbf{0} \quad , \quad \mathbf{r}_\alpha^2 = \mathbf{0}
 \end{array} \right. \quad (4)$$

where  $\mathbf{v}^f$  is the unconstrained velocity predicted by FEM and  $(\mathbf{u}_T^1)^+$  is the positive part of  $\mathbf{u}_T^1$ . We used a modified Delassus operator given by the following expression:

$$\widehat{\mathbf{W}}_{NN} = \mathbf{H}_N \widehat{\mathbf{M}} \mathbf{H}_N \quad , \quad \widehat{\mathbf{W}}_{NT} = \mathbf{H}_N \widehat{\mathbf{M}} \mathbf{H}_T \quad , \quad \widehat{\mathbf{W}}_{TN} = \mathbf{H}_T \widehat{\mathbf{M}} \mathbf{H}_N \quad , \quad \widehat{\mathbf{W}}_{TT} = \mathbf{H}_T \widehat{\mathbf{M}} \mathbf{H}_T \quad (5)$$

Some applications will be presented for belt-pulley systems and for the vibration of a cable subjected to the presence of an obstacle ; Systems are depicted in Figure 1.

## References

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- [3] C. Bertrand, V. Acary, C.-H. Lamarque, and A. Ture Savadkoohi. A robust and efficient numerical finite element method for cables. *Internationa-*

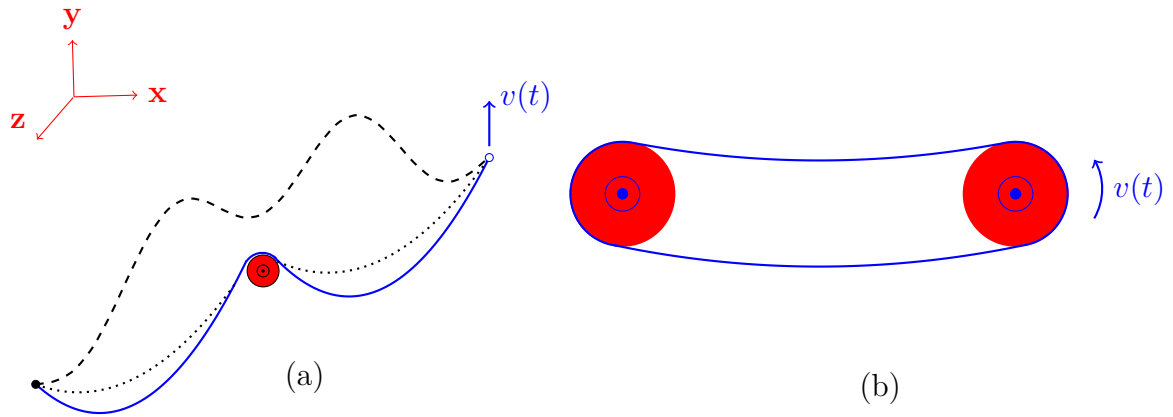


Figure 1: (a) Cable with moving end-support subjected to the presence of a circular obstacle - (b) Belt-pulley system

*tional Journal for Numerical Methods in Engineering*, 121(18):4157–4186, 2020.

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- [5] V. Acary and B. Brogliato. *Numerical Methods for Nonsmooth Dynamical Systems*. Springer-Verlag Berlin Heisenberg, 2008.